

The Basics -Refrigeration Cycle

Jerry Cohen President Jacco & Assoc.

Agenda

- Basic Refrigerant Cycle
- P/h Chart
- Compressor Options
- Condenser Options
- Head Pressure Control
- Thermal Expansion
- Evaporator Options
- Lift
- Evaporative Condensing
- Water Treatment
- Refrigerant Piping







P/h Chart - Isotherm





P/h Chart



p-H Diagram of Refrigeration Cycle



Superheat & Subcooling

Defining Superheat

Superheat is defined as the difference between the temperature at which the refrigerant boils at the given pressure in the evaporator, and the temperature of the refrigerant gas as it leaves the evaporator. In essence, it's how much extra temperature the refrigerant picks up after it has boiled

Heating of a gas, at a constant pressure, above the point at which it has boiled.

Defining Sub-Cooling

Sub-Cooling is defined as the difference between the temperature at which the refrigerant condenses at the given pressure in the condenser, and the temperature of the liquid refrigerant gas as it leaves the condenser. In essence, it's how much extra temperature the refrigerant cools after it has condensed.

Cooling of a liquid, at a constant pressure, below the point at which it was condensed.



P/h Chart



<u>p-H Diagram of Refrigeration Cycle</u> with Sub-Cooling



What is a Compressor?







Scroll Compressor

- Limited to ~ 25 HP
- Tandem Compressor Options
- With or Without VFD









Digital Scroll Compressor

Digital Scroll





Screw Compressor

- Limited to ~ 500 HP
- With or Without VFD







Reciprocating Compressor

• Limited to ~ 100 HP





Centrifugal Compressor

- Virtually Unlimited
- With or Without VFD





Non Positive Displacement compressors raise the pressure and temperature of the refrigerant by converting kinetic energy into pressure.



Centrifugal Compressor

- Oil Free Magnetic Bearing Centrifugal
- With VFD
- Limited to 75 & 150 HP





Magnetic Bearings

- Oil Free Magnetic Bearing Centrifugal
- Limited to 75 & 150 HP





Hot Gas Bypass







Hot Gas Bypass









Condenser – Air Cooled

- Given 95 db/75 wb
- 125 Condensing Temperature





Condenser – Water Cooled

- Given 95 db/75 wb
- 105 Condensing Temperature







Condenser – Evaporative Cooled

- Given 95 db/75 wb
- 105 Condensing Temperature
- Water Cooled WITHOUT the Condenser Pump





The Compressor is a Pump

- Air Cooled
- 35 Suction & 125 Condensing

- Water & Evaporative Cooled
- 35 Suction & 105 Condensing



Refrigerant Pressure/Temperature Chart

Refrigerant Pressure Temperature Chart

Refrigerant

50.9 27. 73.8

52.1 28.6 75.3

53.4 29.5 76.9

54.8 30.4 78.5

56. 31.3 80.2

57.5 32.2 81.7

58.9 33. 83.5

60.3 34. 85.2

61.7 35.0 86.9

63.1 36.0 88.6

64.6 37.0 90.4

66.1 38.0 92.2

67.6 39.0 94.0

69. 40.0

70.6

72.2 42.2 99.5

73. 43.2 101.

75.4 44.3

77. 45.4 105.

106.0 51.3 115.3

116.2 57.4 126.0

127.0 64.0 137.4

138.5 71. 149.3

150.6 78.6 161.9

163 86.7 175.4

177.0

191.3 104.3 204.5

206.4 113.9 220.2

222.3 124. 236.8

239.0 134.9 254.2

256.5 146.3 272.5

274.9 158.4 291.9

294.2

314.5 184.5 333.4

335. 198.3 355.6

357.0

380.9 229.3 403

405. 245.0 429.0

430 262. 456.8

41. 97

95.2

171. 312.

213. 379.

91. 44.3 23.

93. 45.9 24. 67.

95. 47.1 25.3 69.2

97. 48.4 26. 70.1

99.5 49.6 26.9 72.

R-407c R-134a

R-404a

66.2

95

103.3

189.6

Temperature Refrigerant Temperature °C R-22 R-410a R-407c R-134a R-404a R-22 R-410a ٩F °C 51.2 -51.1 -48.3 13 20 -2.2 52.4 -45.6 4.3 18 -1.7 53.7 -42.8 16 54.9 -40.0 0.6 10.1 56.2 -37.2 2.6 13.5 12. 57.5 101.0 -34.4 4.9 17.2 10.3 58.8 103.0 -25 -31.7 7.5 21.4 37 13.5 34 60.2 105.7 -28.9 10.2 25.9 16.8 61.5 107.9 -27.8 11.4 27.8 7.2 18.3 62.9 110. -26.7 12.6 29.7 8.4 19.8 64.3 112.2 -25.6 13.9 31.8 9.5 0.4 21.3 65. 114 -24.4 15.2 33.9 10.7 1.2 22.9 67. 116.3 -23. 16.5 36.1 11.9 24.6 68. 118.9 -22.2 17.9 38.4 13.2 2.8 26.3 70. 121.2 -21. 19.4 40.7 14.6 3.7 28.0 71.5 123.6 42 -20.0 20.9 43.1 15.9 4.6 29.8 73 (125.9 22.4 45.6 17.4 5.5 31.7 74.5 128.3 -18.9 44 -17.8 48.2 18.9 130.3 24.0 33.7 76. -17.3 49.5 19.6 34.7 77.6 24.8 7.0 133.2 50.9 -16.7 25.7 20.4 7.5 35.7 79.2 135.6 52.2 21.2 -16.1 26.5 8.0 36.7 80.8 138.2 8.6 37.7 27.4 53.6 22.0 82.4 140.3 -15. 9.4 -15.0 28.3 55.0 22.8 38.8 84. 143.3 9.1 10.0 -14.4 29.1 56.4 23.7 9.7 39.8 12.8 92.6 156.0 -13.9 30.0 57.9 24.5 10.2 40.9 15.6 101.6 170. -13.3 31.0 59.3 25.4 10.8 42.0 18.3 111.3 185. -12.8 31.9 60.8 26.2 11.4 43.1 21. 121.5 201.5 62.3 -12.2 32.8 27.1 12.0 44.3 23 132.2 218.2 33.8 63.9 -11 28.0 12.6 45.4 26. 143 235 34.8 65.4 29.0 13.2 46.6 155.7 254.6 -11 29 105 Condensing -10.6 35.8 67.0 29.9 13.8 47.8 32. 168.4 274.3 -10.0 36.8 68.6 30.9 14.4 49.0 35 181. 295.0 37.8 70.2 31.8 15.1 50.2 196. 316. -9.4 37 Temperature = 71.9 32.8 15.7 51.5 40. 210.8 339.9 -8.3 39.9 73.5 33.8 16.4 52.7 43.3 226.4 364. -7.8 40.9 75.2 34.8 17.1 54.0 46 242.8 289.6 -7.2 42.0 77.0 35.9 17.7 55.3 260.0 416.4 120 48.9 -6.7 43.1 78.7 36.9 18.4 56.6 51.7 278. 444.5 -6.1 44.2 80.5 38.0 57.9 19.2 130 54.4 297. 474.0 -5.6 45.3 82.3 39.1 19.9 59.3 135 57.2 316. 505.0 -5.0 46.5 84.1 40.2 20.6 60.6 337.4 60.0 537. 140 47.6 85.9 41.3 21.4 62.0 145 62.8 359. 571. 87. 42.4 22. 63.4 381 607

HFC-134a 125 Condensing Temperature = 184.5 PSIG

HFC-410a 125 Condensing Temperature = 444.5 PSIG

🔼 JACCO

HFC-134a

105 Condensing

Temperature =

134.9 PSIG

HFC-410a

339.9 PSIG

Why Head Pressure Control

- Variable Speed Compressors
- Fluctuating Ambient Conditions
- Energy Savings
- Assure Proper TXV Operation



Common Head Pressure Control

- Condenser Fan Cycling
- Variable Speed Condenser Fan
 - Picture Cooling Tower w/ VFD



Low Ambient Head Pressure Control

Flooded Condenser







Capillary Tube, Fixed Orifice, Thermal Expansion

• Cheap but Effective





Pressure Thermal Expansion

Old and Stable Technology







Electronic Thermal Expansion

• Very Controllable











Evaporator – Air Cooled

- Given 55 Degree LAT
- 45 Degree Suction Temperature





Evaporator – Chilled Water

- Given 55 Degree LAT
- 45 Degree Chilled Water/Glycol
- 35 Degree Suction Temperature





Heat Pump Refrigeration Cycle





Pump Out vs. Pump Down

 The purpose of Pump Down and Pump Out is to minimize the effects of migration, which can result in a flooded start on the next startup.



Pump Out

 Also known as a "one time pump out", is conducted by closing a solenoid in the liquid line while the compressor is still running, all of the refrigerant in the system downstream of the solenoid is removed. The compressor is shutdown by a low pressure control, and remains off until the next call for cooling which opens the solenoid valve and closes the contacts for the compressor. Thus the compressor pumps out the system after each running cycle, then remains off until the next call for cooling.



Pump Down

 Also known as "continuous pump down", is conducted exactly the same way as pump-out with the exception that the call for cooling ONLY controls the solenoid. Thus, during a normal running cycle, when demand is met, the solenoid is closed. The compressor pumps out all of the refrigerant downstream of the solenoid then shuts of on a low pressure control. The difference being that the compressor is allowed to restart any time the low pressure control makes, thus keeping the suction side of the system pumped down continuously. This usually comes into play if there is a leaking pump down solenoid.



Accumulators and Receivers

- A vessel for holding refrigerant liquefied by the condenser.
- A vessel for preventing liquid slugging of the compressor.



Condenser/Evaporator Balance





Refrigerant Piping









Cooling with MHGRH



Determining Loads for Line Sizing

	Dual Com	pressor Un	its	
(Two In	dependent	: Refrigerai	nt Circuits)	
Condensing Unit Total Capacity (Tons)	Suction Line (Tons)	Liquid Line (Tons)	Reheat Line (Tons)	Hot Gas Bypass Line (Tons)
100%	50%	50%	50%	50%



Dua (Two In	l Tandem (dependent	Compresso t Refrigerai	r Units nt Circuits)	
Condensing Unit Total Capacity (Tons)	Suction Line (Tons)	Liquid Line (Tons)	Reheat Line (Tons)	Hot Gas Bypass Line (Tons)
100%	50%	50%	50%	25%

Suction Line

 Suction Line – Vertical UP, Vertical Down, or Horizontal Minimum Velocity = 1000 fpm Maximum Velocity = 4000 fpm Maximum Line Loss = 6F

2. Suction Line Flow – Vertical UP

Tonnage > Minimum Tonnage Required for Oil Return* Velocity > Minimum Velocity Required for Oil Return* Trap Every 10 Feet along Vertical-Up Riser

*For oil to flow up along a pipe wall, a minimum drag of gas flow (friction gradient) is required. ECat uses the following data to ensure oil is returned up vertical suction lines.

 Line Size

 2 in. or less
 Above 2 in.

 0.35 psi/100 ft
 0.20 psi/100 ft

For tandem compressors and variable capacity compressors the oil return must work with one compressor operating with reduced capacity. So, the vertical line must be selected to return oil at minimum flow. This can result in too high a pressure drop at full capacity.



Suction Line – Variable Capacity Compressors

Rule 1 – The oil MUST BE returned to the compressor, there is no rule 2.

- 1. Size the vertical portion of the suction line to maintain the minimum velocity for oil return at minimum load and suffer the pressure drop penalty at full capacity. Refrigerant and oil must not log in the line when it is active.
- 2. For long vertical lines provide traps, size the vertical suction line for minimum load and provide traps to accumulate oil when the refrigerant line is idle, not when it is active. Make sure the compressor can handle the slug oil when restarted, an accumulator may be required.
- 3. To maintain oil return at the minimum load <u>without a pressure drop</u> <u>penalty</u> use a double suction riser.



Double Suction Riser



Fig. 3 Double-Suction Riser Construction

- 1. Two vertical lines operate in parallel at the full load to return suction gas and oil to compressor. The pressure drop in riser A and B are equal when both have gas flow. The diameter of riser A is always less than riser B.
- 2. At reduced load the velocity in riser B drops to the point where it can not return oil. The trap on riser B fills with oil and seals off the gas flow in riser B and all gas flows thru riser A. Riser A must be sized to return oil at minimum flow.



Liquid Line Sizing Rules

Liquid Line – Vertical Down, or Horizontal - Maximum Line Loss 6F

Liquid Line – Vertical Up

Maximum Line Loss 8F Including Loss, accessories (LLSV, Filter/Dryer) and Liquid Lift The TXV is rated for 100% liquid at the inlet (no flash gas) Every foot of vertical lift = -.43 psi or -.095 F sub-cooling for R410A. For proper operation of the system the sub-cooling at the TXV inlet should be at least 2F.



Hot Gas Line Sizing Rules

HGB Line - Vertical UP, Vertical Down, or Horizontal Size as discharge line for Single Compressor with SST = 32, SDT = 90

HGB Line – Vertical UP

Trap Every 10 Feet along Vertical-Up Riser Drain line in accordance AAON piping guidelines.

Reheat Line = Vertical UP, Vertical Down, or Horizontal Minimum Velocity = 900 fpm Maximum Velocity = 3000 fpm Maximum Line Loss = 10F

Reheat Line – Vertical UP

Tonnage > Minimum Tonnage Required for Oil Return Velocity > Minimum Velocity Required for Oil Return Trap Every 10 Feet along Vertical-Up Riser Drain line in accordance AAON piping guidelines.





Heat Pump Suction/Discharge Line

Rule 1 – The oil MUST BE returned to the compressor, there is no rule 2.

Economics, pressure drop, noise, and oil entrainment establish feasible design velocities in refrigerant lines (ASHRAE Table 1 is a guide).

Table 1	Recommended Gas Line Velocities
Suction line	900 to 4000 fpm
Discharge line	2000 to 3500 fpm





Hot Gas Bypass

HGBP	Lead	and	Lag
------	------	-----	-----

Condensing Unit Options Tagging: Split CU # 1 Voltage: 230V/3Ø/60Hz Compressor Type: R-410A Tandem Scroll Comp

•

•

tion Liqu	iid Rehe	at			
Elbow Qu Line Leng Vertical Li	antity: th: ft:	4 3 1		Suction Line Flow:	Down
		Suct	ion Line S	election	
Pipe OD	Equiv. Length	Temp. Loss(F)	Vel (fpm)	Min. Tons For Oil Return	Qty. of Req. Traps
1.125	36.8	2.34	3928	2.38	0
1.375	39.2	0.94	2663	4	0
1.625	40.4	0.41	1882	6.33	0
	100	0.1	1047	10.70	0





Modulating Hot Gas Reheat ✓ Heat Pump

Hot Gas Bypass Hubby I and HGBP Lead and Lag

i ayyıny.	
Split CU # 1	
Voltage:	
230V/3Ø/60Hz	
Compressor Type:	

(i unp/ s	uction L	iquid Re	heat			
Elbow Qu	antity:	4		Suction Line Flow:	Down	
Line Length: Vertical Lift:		3	0			
		1	0			
		Heat Pump	/Suction I	Line Selection		
	Equiv.	Temp. Loss(F)	Vel (fpm)	Min. Tons For Oil Return	Qty.of Req.Traps	
Pipe OD	Length				1/2/	
Pipe OD 1.125	47	2.99	3936	2.44	2	



Important Functions

- **Perform System Rating** at the balance point of the Evaporator, Compressor and Condenser for a given Ambient and Evaporator Entering Conditions.
- Rate and allow selection of interconnecting piping including:
 - Suction Line
 - Discharge Line (heat pump heating mode)
 - Liquid Line
 - HGB Line
 - Reheat Line
- Rate the system with the selected line sizes.



In other words, combine 4 programs

- 1. AHU Rating Program
- 2. CU Rating Program
- 3. EES Refrigerant Line Sizer
- 4. System Balance Program



	Second Piping Cal	Culator Autoprantis (MIDA) Tora (10) (MIDA) Muna + 8.100 (MIDA) Muna + 8.100 (MIDA) Muna + 8.100 (MIDA)	AAON Inc. Reference have been false, cancers from the ans too ball to the estimate	
Partial Partial 1(2 m - 2) Marcine 2014 1(2 m - 2) Anno (2 m - 2) 1(2 m - 2) Anno (2 m - 2) 1(2 m - 2) Anno (2 m - 2) 1(2 m - 2) Anno (2 m - 2) 1(2 m - 2) Anno (2 m - 2) 1(2 m - 2) Anno (2 m - 2) 1(2 m - 2) Anno (2 m - 2) 1(2 m - 2) Anno (2 m - 2) 1(2 m - 2) Anno (2 m - 2) 1(2 m - 2) Anno (2 m - 2) 1(2 m - 2) Anno (2 m - 2) 1(2 m - 2) Anno (2 m - 2) 1(2 m - 2) Anno (2 m - 2)	Section Line (First 30) Her, 12(3) Her, 12(3) Her, 12(3) How, 12(3)	Lighting (1) and (2) (4) a scalar by (4) a scalar by	Mitte Inneigi Inneidi Inneidianeidianeidi Inneidianeidi Inneidianeidi Inneidianeidi Inneidianeidi Inneidianeidi Inneidianeidi Inneidianeidi Inneidianeidi Inneidianeidi Inneidianeidi In	
SCT _{ME} = 16.6 (F) STT _{ME} = 22.6 (F) Termine = 22.6 (F)	BCF _{WAR} + 103.4 (pring) 15P _{Hall} = 104.8 (pring)	Countrypeb) Treger	ariyeene 3	







Refrigerant Line Sizing Rules

Another Rule 1 Refrigerant lines are never horizontal and should always slope (1/8" per foot) in direction of flow.





Refrigerant Piping

All piping must pitch from the top of the suction riser down to the condenser without any additional lifts or traps. The fall needs to be 1/8th per foot. The suction riser must be straight up with no 45's or elbows. Suction line should be 1 5/8OD. Elbows to be long radius. All existing system traps other than suction riser should be eliminated from the suction line. Liquid line can be left alone.







Break